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METHOD AND APPARATUS FOR EJECTING A
REMOVABLE DATA STORAGE CARTRIDGE

TECHNICAL FIELD OF THE INVENTION

This invention relates in general to a data storage cartridge that can be removably inserted into a drive and, more particularly, to a method and apparatus for effecting ejection of the cartridge from the drive.

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BACKGROUND OF THE INVENTION

Computer technology has evolved very rapidly over the past 25 years. One aspect of this evolution has been a progressively increasing demand for progressively more storage capacity in removable data storage cartridges. For example, floppy disks capable of storing approximately 360 KB of data gave way to floppy disks capable of storing approximately 720 KB, which in turn gave way to floppy disks capable of storing approximately 1.44 MB of data.

Thereafter, removable data storage cartridges with still higher capacities became commercially available, for example in the form of cartridges available under the tradename ZIP from Iomega Corporation of Roy, Utah, which is the assignee of the present application. ZIP cartridges provided data storage capacities on the order of 100MB to 250MB. Still another significant increase in storage capacity was subsequently realized when Iomega introduced removable cartridges under the tradename JAZ, which have storage capacities on the order of 1 GB to 2 GB. Nevertheless, the demand for still greater storage capacities in removable cartridges continues to progressively increase, such there is a current need for cartridges capable of storing 5 GB to 20 GB, or even more.

Existing removable cartridges of the type discussed above tend to be relatively small and lightweight, and tend to consist primarily of a housing with a magnetic disk rotatably supported therein. The drive which receives the cartridge includes a head cooperable with the disk in the cartridge. Further, the drive typically includes some form of eject mechanism for facilitating

removal of the cartridge. While these existing eject mechanisms have been generally adequate for their intended purposes, they have not been satisfactory in all respects, and are more suitable for preexisting cartridges than the newest generation of cartridges.

In this regard, next-generation cartridges which are currently in development will be more complex than preexisting cartridges, and will be somewhat larger and heavier. They will include a sealed housing containing not only a rotatable storage medium in the form of a hard disk, but also a motor for rotating the disk, at least one read/write head, and a mechanism for effecting movement of the read/write head. Connectors on the cartridge and drive will be releasably engaged in order to electrically couple a number of electrical signals between the cartridge and drive. Pre-existing cartridges do not use connectors of this type. Given various differences of this type, pre-existing eject mechanisms are not well-suited for use with these next generation cartridges.

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SUMMARY OF THE INVENTION

From the foregoing, it may be appreciated that a need has arisen for a method and apparatus for facilitating ejection from a drive of a removable data storage cartridge, where the drive and the cartridge have cooperating connectors. According to the present invention, a method and apparatus are provided to address this need, and relate to operation of an apparatus which includes a receiving unit having a connector and having a cartridge receiving portion into which a cartridge can be removably inserted to an inserted position. The method and apparatus involve: supporting a locking part for movement between locking and unlocking positions; causing the locking part, when a cartridge is disposed in the cartridge receiving portion, to engage the cartridge in the locking position thereof so as to resist cartridge removal from a position in which a connector of the cartridge is engaged with the connector of the receiving unit for the purpose of electrically coupling the receiving unit to an information storage section of the cartridge; resisting movement of the locking part away from the locking position with a retaining portion when a cartridge is disposed in the cartridge receiving portion and is engaged with the locking part; supporting an eject member for movement between a retracted position and an eject position; causing the eject member to move the locking member away from the locking position toward the unlocking position during movement of the eject member from the retracted position to the eject position; causing the eject member, when a cartridge is disposed in the cartridge receiving portion, to engage and move the cartridge away from its insertion position as the eject

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5 member moves from the retracted position to the eject position, in a manner effecting disengagement of the connector of the receiving unit from a connector of the cartridge; and responding to the occurrence of a predetermined event by causing a drive portion to effect movement of the eject member from the retracted position to the eject position.

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BRIEF DESCRIPTION OF THE DRAWINGS

FIGURE 1 is a diagrammatic perspective view showing an information storage device which embodies the present invention;

5 FIGURE 2 is a diagrammatic perspective view of an interface module which is a component of the information storage device of FIGURE 1;

10 FIGURE 3 is a diagrammatic perspective view of a drive module which is a component of the information storage device of FIGURE 1;

FIGURE 4 is a diagrammatic side view of the drive module of FIGURE 3;

FIGURE 5 is a diagrammatic rear view of the drive module of FIGURE 3;

15 FIGURE 6 is a diagrammatic top view of the drive module of FIGURE 3;

FIGURE 7 is a diagrammatic perspective view of a cartridge which is a component of the information storage device of FIGURE 1;

20 FIGURE 8 is a diagrammatic fragmentary front view of the drive module of FIGURE 3, with certain structure omitted for clarity, including a front portion of a housing thereof;

25 FIGURE 9 is a diagrammatic perspective view showing a back and a side of a pawl which is a component of the drive module of FIGURE 3;

FIGURE 10 is a diagrammatic perspective view showing a front and a side of the pawl of FIGURE 9;

30 FIGURE 11 is a diagrammatic perspective view showing front and top sides of an eject member which is a component of the drive module of FIGURE 3;

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FIGURE 12 is a diagrammatic perspective view of the eject member of FIGURE 11, showing bottom and front sides thereof;

FIGURE 13 is a diagrammatic exploded perspective view of the portion of the drive module which is depicted in FIGURE 8;

FIGURE 14 is a diagrammatic fragmentary front view similar to FIGURE 8, but showing the cartridge of FIGURE 7 inserted part way into the drive module;

FIGURE 15 is a diagrammatic fragmentary front view of the drive module and cartridge which is similar to FIGURE 14, except that the cartridge has been fully inserted, various movable parts of the drive module are in different operational positions, a front portion of the housing of the cartridge has been removed, and certain circuitry and information storage structure provided within the cartridge have been omitted for clarity;

FIGURE 16 is a diagrammatic fragmentary front view of an alternative embodiment of the drive module of FIGURE 3, which is similar to a portion of FIGURE 8 depicted in an enlarged scale, but showing a switch which is not present in the embodiment of FIGURE 8;

FIGURE 17 is a diagrammatic fragmentary perspective rear view of a portion of an alternative embodiment of the drive module of FIGURE 3; and

FIGURE 18 is a diagrammatic perspective view showing rear and bottom sides of an eject member which is an alternative embodiment of the eject member shown in FIGURES 11 and 12, and which is a component of the drive module shown in FIGURE 17.

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DETAILED DESCRIPTION OF THE INVENTION

FIGURE 1 is a diagrammatic perspective view showing an information storage device 10 which embodies the present invention, and which can be coupled by a cable 12 to a host computer system that is not illustrated. The information storage device 10 includes a receiving unit or cradle 13, and includes an information storage cartridge 14 which is removably inserted into the cradle 13. The cartridge 14 is inserted into and removed from the cradle 13 in directions which are almost vertical, as indicated by a double-headed arrow 16.

The cartridge 14 has a housing, and includes within the housing a not-illustrated information storage media, which in the disclosed embodiment is a rotatable hard disk of a known type. The cartridge 14 also includes within the housing some not-illustrated structure and circuitry that facilitates reading and writing of information from and to the hard disk, including a spin motor for effecting rotation of the disk, a read/write head supported for movement adjacent a surface of the disk, and circuitry coupled to the read/write head. A detailed comprehension of these internal components of the cartridge is not needed in order to understand the present invention, and they are therefore not illustrated and described here in detail.

The cradle 13 includes a base or interface module 17 and a drive module 18, which are physically separate modules that are releasably coupled to each other in a manner described in more detail later. Two manually operable release buttons 22 are provided on opposite sides of the drive module 18, although only one of these two buttons 22 is visible in FIGURE 1. When the two

release buttons 22 are simultaneously manually pressed, the detachable coupling between the drive module 18 and the interface module 17 is released, so that they can be separated.

5 The interface module 17 has a window 23 provided through a front wall portion thereof. A liquid crystal display (LCD) 26 is provided on the drive module 18, and is visible through the window 23 of the interface module 17 when the two modules are releasably coupled to each other. A manually operable eject button 27 is provided on the interface module 17. When the eject button 27 is manually pressed downwardly, the interface module 17 sends the drive module 18 an electrical signal which causes the drive module 18 to release a locking mechanism which holds the cartridge in place, and to then effect a partial ejection of the cartridge 14, in a manner described in more detail later.

10 The drive module 18 has an opening through a front wall thereof, in which is mounted a magnifying lens 28. When the cartridge 14 is removably inserted in the cradle 13, a label on the cartridge can be viewed through the lens 28.

15 FIGURE 2 is a diagrammatic perspective view of the interface module 17 by itself. The interface module 17 has a housing with a top surface 37, and has a recess 36 which opens vertically downwardly into the interface module from the top surface 37 thereof. Two smaller recesses 41 and 42 are provided on opposite sides of the recess 36, and are used to facilitate the detachable coupling of the interface module 17 to the drive module 18, in a manner described later. Immediately behind the recess 36, the top surface 37 has a further recess 46,

which has therein an upwardly facing bottom surface 47. The top surface 37 of the housing also has two bosses 48 and 49 disposed on opposite sides of the recess 36, at locations spaced forwardly from the recesses 41 and 42.

5 A connector 52 is provided within the recess 36, at a location spaced rearwardly from the front wall portion that has therein the window 23. The interface module 17 has circuitry therein which is coupled to the connector 52 and which is responsive to operation of the eject
10 button 27. However, details of the circuitry within the interface module 17 are not needed in order to understand the present invention, and the circuitry is therefore not illustrated and described in detail.

FIGURE 3 is a diagrammatic perspective view of the
15 drive module 18 by itself. In addition to the structure of the drive module 18 which has already been described above, it can be seen in FIGURE 3 that the drive module 18 has a projection 61 which extends downwardly beyond the bottom of the housing of the drive module 18. The
20 projection 61 has the LCD display 26 mounted on a front side thereof.

The drive module 18 has two pawls 62 and 63 on opposite sides of the projection 61. The two pawls 62 and 63 are each supported for limited movement toward and
25 away from the projection 61, and are each biased to move in a direction away from the projection 61 by a respective spring which is not illustrated. The two release buttons 22 on opposite sides of the drive module 18 are each operatively coupled to a respective one of
30 the pawls 62-63. When the two release buttons 22 are simultaneously manually pressed, they each effect movement of a respective one of the pawls 62-63 in a

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direction inwardly toward the projection 61, against the urging of the springs that bias the pawls outwardly.

FIGURE 4 is a diagrammatic side view of the drive module 18. FIGURE 4 shows a connector 67 which extends
5 downwardly beyond the housing of the drive module 18, at a location spaced rearwardly from the projection 61. As evident from FIGURE 4, the connector 67 does not extend downwardly as far as the projection 61.

FIGURE 5 is a diagrammatic rear view of the drive
10 module 18. The release buttons 22 on opposite sides of the drive module 18 are both clearly visible in FIGURE 5. Further, FIGURE 5 shows that the housing of the drive module 18 has a downward projection 69 on the rear side of its bottom surface.

When the drive module 18 is to be releasably coupled
15 to the interface module 17, the projection 61 on the drive module 18 is inserted downwardly into the recess 36 (FIGURE 2) in the interface module 17, in the region between the connector 52 and the front wall portion with
20 the window 23. The projection 61 cooperates with surfaces in the recess 36 in order to help guide the connector 67 on the drive module 18 into alignment with and then engagement with the connector 52 on the interface module 17.

As the drive module 18 continues to be moved
25 downwardly relative to the interface module 17, and after the connectors 67 and 56 have established electrical contact, the downward projection 69 (FIGURE 5) on the drive module 18 engages the bottom surface 47 of the
30 small recess 46 in the interface module 17. Meanwhile, the bosses 48 and 49 on the interface module 17 engage respective locations on the underside of the housing of

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the drive module 18. The engagement of the bosses 48-49 with the underside of the housing of the drive module 18, in conjunction with the engagement of the projection 69 with the surface 47, establishes three points of contact which define and maintain a proper orientation of the drive module 18 relative to the interface module 17.

As the drive module 18 is being moved downwardly into engagement with the interface module 17, the pawls 62 and 63 respectively enter the recesses 41 and 42, and are each temporarily urged in an inward direction due to their engagement with the edges of the recesses 41-42 during the insertion movement. As the drive module 18 reaches its final position, the pawls 62-63 reach a position where they can move outwardly under the resilient urging of the not-illustrated springs, so that they releasably lockingly engage the recesses 41-42 in a manner that releasably couples the drive module 18 to the interface module 17.

In order to subsequently release the drive module 18 from the interface module 17, the two release buttons 22 are simultaneously manually pressed in order to move the pawls 62 and 63 inwardly, thereby releasing the locking engagement between the pawls 62-63 and the recesses 41-42. This in turn permits the drive module 18 to be lifted approximately vertically upwardly out of engagement with the interface module 17.

FIGURE 6 is a diagrammatic top view of the drive module 18, looking down into a vertical recess 76 that can removably receive the cartridge 14 (FIGURE 1). On opposite sides of the recess 76 are two guide rails 77 and 78, which extend approximately vertically down into the recess 76 lengthwise thereof, and which project

inwardly into the recess 76 from opposite sides thereof. The guide rails 77-78 are each slightly offset from the center of the recess 79 in a direction normal to an imaginary plane extending between the guide rails.

5 At the bottom of the recess 76 is a connector 73, which is shown diagrammatically in broken lines in FIGURE 6. On opposite sides of the connector 83 are two movably supported latching pawls 81 and 82. The pawls 81 and 82 are each shown diagrammatically by broken lines in FIGURE 6, and are each supported for limited movement toward and
10 away from the connector 83. The pawls 81-82 and the connector 83 are discussed in more detail later.

FIGURE 7 is a diagrammatic perspective view of the cartridge 14 by itself. The cartridge 14 has on one side
15 of its housing 85 a label 86, which carries indicia that is not shown in the drawings. When the cartridge 14 is removably inserted into the cradle 13, as shown in FIGURE 1, the indicia on the label 86 is visible through the magnifying lens 28. The cartridge housing 85 has at
20 one end two spaced outward projections 87 and 88. The housing 85 has on opposite sides thereof two elongate grooves 91 and 92. The grooves 91 and 92 each extend approximately half the length of the cartridge 14, beginning from the end surface of a respective one of the
25 projections 87 and 88. As evident in FIGURE 7, the sidewalls of the grooves 91-92 are flared slightly at the ends of the grooves adjacent to the projections 87-88. Also, the grooves 91 and 92 are each offset slightly with
30 respect to the center of the cartridge 14, in a direction normal to an imaginary plane extending between the grooves 91-92.

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Between the projections 87 and 88, in an end surface of the cartridge 14, is a connector 93. On opposite sides of the connector 83, the housing 85 of the cartridge 14 has two openings or recesses 96 and 97.

5 With reference to FIGURES 6 and 7, the guide rails 77-78 and the slots 91-92 ensure that there is only a single orientation in which the cartridge 14 can be inserted into the recess 76. In particular, since the grooves 91-92 only extend approximately half the length
10 of the cartridge 14, it is not possible to insert the wrong end of the cartridge 14 very far into the recess 76, because the guide rails 77-78 will engage an end surface of the cartridge 14 and thereby prevent further insertion of the cartridge 14 with that orientation.

15 Moreover, even when the correct end of the cartridge 14 is introduced into the recess 76, the cartridge 14 must be oriented so that the label 86 on the cartridge 14 is facing in the same direction as the lens 28 on the drive module 18. This is because, as discussed above,
20 the guide rails 77-78 are offset slightly with respect to a center of the recess 76, and the grooves 91-92 are offset slightly with respect to a center of the cartridge 14. If the label 86 is facing in a direction opposite from the direction in which the lens 28 is facing, there
25 will be a mechanical interference between the guide rails 77-78 and the end surfaces of the projections 87-88, which will occur after the cartridge has been only partially inserted, so as to prevent any further insertion movement of the cartridge 14 into the recess
30 76.

When the cartridge 14 is inserted into the recess 76 with the proper orientation, the flared side surfaces at

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5 First, their cooperation ensures that the connectors
93 and 83 will be accurately aligned as they move into
engagement with each other. Second, their cooperation
has the effect of positioning the cartridge 14 within the
recess 76 in a manner so that the exterior surfaces of
10 the cartridge 14 are spaced from and do not rub against
the internal surfaces of the recess 76, except to the
extent that surfaces on the guide rails 77-78 engage
surfaces in the grooves 91-92. Thus, even after the
cartridge 14 has been inserted into and removed from the
15 drive module 18 many times, most of the exterior surfaces
of the cartridge 14 will still look very new, rather than
being highly scuffed.

20 mating engagement, the pawls 81 and 82 of the drive
module 18 respectively move into the recesses 96 and 97,
and are temporarily deflected inwardly by edges of the
recesses as they enter the recesses. Then, as the
connectors 83 and 93 reach proper mating engagement, the
25 pawls 81 and 82 reach positions where they move outwardly
so that locking edges thereon engage edges of the
recesses 96-97 in a manner which prevents manual
withdrawal of the cartridge 14 from the drive module 18.

30 module 18 in a normal manner, an operator manually
presses the eject button 27 on the interface module 17
(FIGURES 1 and 2). The not-illustrated circuitry within

the interface module 17 will respond to actuation of the eject button 27 by transmitting an electrical signal through the connectors 52 and 67 to the drive module 18. The drive module 18 has a release mechanism which then moves the pawls 81 and 82 inwardly toward the connector 83, until they are released from latching engagement with the recesses 96 and 97. The release mechanism then moves the cartridge 14 upwardly a sufficient distance relative to the drive module 18 so as to effect disengagement of the connector 93 from the connector 83. This release mechanism is configured to alternatively permit a manually-operated release that moves the pawls and separates the connectors, for example in a situation where the motor which drives the release mechanism does not currently have power, or experiences some type of failure. This release mechanism within the drive module 18 will be described in more detail later.

FIGURE 8 is a diagrammatic fragmentary front view of the drive module 18 with a front portion of the housing thereof removed, so that only a rear portion 101 of the drive module housing is visible. The drive module 18 is shown in FIGURE 8 with no cartridge 14 inserted therein. For clarity, and with reference to FIGURES 3 and 4, FIGURE 8 does not show the pawls 62-63, the release buttons 22 that operate them, or other associated structure. However, it will be noted that the housing portion 101 has recesses 106 and 107 through which the pawls 62-63 project out of the housing, and also has recesses 108 and 109 through which the release buttons 22 project out of the housing.

The connector 83 of the drive module 18, which can electrically couple the drive module 18 to a cartridge

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14, is shown in approximately the center of FIGURE 8. It is fixedly supported on a circuit board 112, which in turn is fixedly secured on the housing portion 101. The connector 83 thus does not move relative to the housing portion 101. The pawls 81 and 82 are shown on opposite sides of the connector 83, and each pawl has one end pivotedly supported on a respective pivot pin 116 or 117, the pivot pins 116-117 each being provided on the housing portion 101.

A U-shaped wire spring 119 has two spaced legs coupled by a bight, the bight being supported by three posts 121-123 which are each an integral part of the housing portion 101. The post 122 has a transverse slot in the side thereof facing the connector 83, and this slot receives the center of the bight of the spring. The posts 121 and 123 are each provided at an inside corner of the spring, between the bight and a respective one the legs. Each of the legs of the spring 119 engages a respective one of the pawls 81 and 82 on a side thereof nearest the connector 83, and resiliently urges the associated pawl to pivot in a direction away from the connector 83.

In the disclosed embodiment, the pawls 81 and 82 are mirror images of each other, but they could alternatively be identical. Therefore, only the pawl 81 is illustrated and described in detail. More specifically, FIGURE 10 is a diagrammatic perspective view showing a front and side of the pawl 81, and FIGURE 9 is a further diagrammatic perspective view showing a back and side of the pawl 81. The pawl 81 has upper and lower portions 131 and 132, which extend at an obtuse angle to each other. A cylindrical opening 133 extends transversely through the

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pawl 81 at the lower end of the portion 132, and pivotally receives the pivot pin 116 (FIGURE 8). On the back side of the upper portion 131 are three bosses 136-138, which are arranged so that a slot is effectively defined between the boss 136 and the bosses 137-138. An end of the wire spring 119 is received in this slot, and the bosses 136-138 maintain the end of the spring in proper operational engagement with the pawl 81.

The upper end of the pawl 81 is an irregular pyramid-shaped frustrum, with an upwardly tapering cross section, and includes three inclined surfaces 141-143 that taper to an approximately square top surface 144. The inclined surfaces 141-143 help guide the upper end of the pawl into a respective one of the recesses 96-97 (FIGURE 7) in the cartridge 14. The front surface of the upper portion 131 of the pawl 81 includes two spaced surface portions 146 and 147. A recess 148 is provided between the surface portions 146 and 147, and a further recess 149 is provided on an opposite side of the surface portion 147 from the recess 146.

The recess 148 has a downwardly facing surface portion 151, a forwardly facing surface portion 152, and an upwardly facing surface portion 153. The surface portion 152 is approximately perpendicular to each of the surface portions 151 and 153. A further surface portion 154 is provided between the surface portions 153 and 147, and is angled to extend at approximately 45° with respect to each of the surface portions 153 and 147.

The recess 149 includes at its lower end an upwardly facing stop surface 156. It will be noted that the stop surface 156 is disposed approximately at the intersection of the upper and lower portions 131 and 132 of the pawl

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81. A forwardly facing surface portion 157 extends upwardly within the recess 149 from the stop surface 156, approximately perpendicular to the stop surface 156. An inclined surface portion 158 extends between the surface portions 157 and 147, at an angle of approximately 45° with respect to each. The surface portions 142 and 146 together serve as an upper cam surface, and the surface portions 147, 158 and 157 together serve as a lower cam surface. The recess 148 is disposed between these two cam surfaces, and the surface portion 154 provides a transition between the recess 148 and the lower cam surface, for a purpose discussed later.

Referring again to FIGURE 8, a lift or eject member 171 is supported for reciprocal vertical movement within the drive member 18. The manner in which the eject member 171 is movably supported will be described in more detail below, but first the eject member itself is described in detail, with reference to FIGURES 11 and 12.

More specifically, FIGURE 11 is a diagrammatic perspective view of the eject member 171, showing the front and top sides thereof, and FIGURE 12 is a further diagrammatic perspective view of the eject member 171, showing the bottom and front sides thereof. The eject member 171 has a vertical portion 173 and a horizontal portion 174, which are each approximately platelike. The horizontal portion 174 extends forwardly from the lower end of the vertical portion 173, approximately perpendicular thereto.

The vertical portion 173 has vertically extending edge portions 176 and 177 on opposite sides thereof, which facilitate support of the eject member 171 for vertical sliding movement within the drive module 18, in

a manner described later. A central portion 178 of the vertical portion 173 has a downwardly facing surface 181, which is substantially centered between the edge portions 176 and 177, and which serves a purpose described later.

5 At the upper end of the central portion 178 is a rearwardly extending guide pin 183, which also facilitates the vertical sliding movement of the eject member 171 within the drive module 18, in a manner discussed later.

10 Turning to the horizontal portion 174 of the eject member 171, a central opening 186 of approximately rectangular shape extends vertically through the horizontal portion 174. As best seen in FIGURE 12, a bevel 187 extends completely around the opening 186 on
15 the lower side of the horizontal portion 174. The bevel 187 forms an angle of approximately 45° with respect to the bottom surface of the horizontal portion 174, and also with respect to the vertically extending edge surfaces of the central opening 186.

20 Two ridges 191 and 192 project upwardly from the upper side of the horizontal portion 174, adjacent opposite ends of the opening 186. The ridges 191-192 extend parallel to each other in a front to rear direction, and are positioned so that each has a side
25 which faces the other thereof and which is approximately flush with the associated end surface of the opening 186. The ridges 191 and 192 each have a 45° bevel surface 193 or 194 between the top thereof and the side thereof which faces the other ridge. The ridges 191-192 increase the
30 effective thickness of the horizontal portion 174 at each of the ends of the central opening 186, for a purpose discussed later. The two ends of the central opening

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186, including surface portions on the ridges 191-192 and also on the horizontal portion 174, serve as respective cam followers, in a manner discussed later. Two circular openings 196 and 197 extend vertically through the horizontal portion 174, on opposite sides of the central opening 186.

In order to explain how the eject member 171 is slidably supported within the drive module 18, reference is made to FIGURE 8, and also to FIGURE 13. FIGURE 13 is a diagrammatic exploded perspective view of the portion of the drive module 18 which is depicted in FIGURE 8. FIGURE 13 shows that the housing portion 101 has on one side thereof an elongate surface 201 that extends vertically and faces forwardly, an upwardly facing surface 202 near the lower end of surface 201, and a downwardly facing surface 203 near the upper end of surface 201. A surface 204 extends upwardly along the surface 201 perpendicular thereto, and faces sidewardly.

A side member 206, which has the guide rail 78 thereon, is secured by not-illustrated screws to the housing portion 101, in a position so that a rearwardly facing surface 207 on the side member 206 is spaced slightly from and faces the forwardly facing surface 201 on the housing portion 101. As a result, the surfaces 201 and 207 effectively define opposite sides of a slot, the surface 204 effectively defines the bottom of the slot, and the surfaces 202 and 203 effectively define ends of the slot. The outer edge 177 of the eject member 171 is vertically slidably received within this slot.

The end surfaces 202 and 203 are capable of engaging the eject member 171 to limit its vertical movement, but there is other structure which limits upward and downward

movement of the eject member 171, such that it never actually engages the surfaces 202 and 203 during normal operation. The opposite outer edge portion 176 of the eject member 171 is slidably supported in a similar slot defined by structure which is a mirror image of the structure that has just been described, including a further side member 211 which is a mirror image of the side member 206, and which has thereon the guide rail 77. The housing portion 101 has a vertically extending slot 213 which slidably receives the guide pin 183 of the eject member 171.

As best seen in FIGURE 8, the connector 83 projects through the central opening 186 provided in the horizontal portion of the eject member 177, without touching the eject member 171. The upper portions 131 of the two locking pawls 81 and 82 each extend through the central opening 186, and the lower cam surfaces on the pawls can each slidably engage one of the ends of the central opening 186, which as mentioned above can function as cam followers. The cooperation between the pawls and the cam followers on the eject member 171 will be described in more detail later.

The drive module 18 includes a motorized drive mechanism 221, which is best seen in FIGURE 8. The drive mechanism 221 includes a vertical support plate 222, which is fixedly secured by not-illustrated screws to the housing portion 101. The support plate 222 has an electric motor 223 mounted thereon, which in the disclosed embodiment is a direct current brush motor. The motor 223 has a rotatable shaft 224, and a worm gear 226 is mounted on the shaft 224.

Three spur gears 227, 228 and 229 are rotatably supported on respective pivot pins provided at spaced locations on the support plate 222. The worm gear 226 drivingly engages teeth provided along the outer periphery of the gear 227. Integral with the gear 227 is a concentric further spur gear 231 of substantially smaller diameter, which drivingly engages teeth provided on the outer periphery of the gear 228. Still another spur gear 232 is concentric to and integral with the gear 228, but of substantially smaller diameter. The spur gear 232 engages teeth provided along the outer periphery of the gear 229, which is referred to herein as a crank gear.

The crank gear 229 has, at a location eccentric to its pivot, a forwardly projecting actuating pin 236. When the drive mechanism 221 is idle, the motor 223 is off and the crank gear 229 is in the angular position shown in FIGURE 8. When the drive mechanism 221 is actuated, the motor 223 is energized in a manner which causes the crank gear 229 to rotate 360° in a clockwise direction in FIGURE 8, so that the actuating pin 236 begins from and ends back up in the position shown in FIGURE 8.

FIGURE 14 is a diagrammatic fragmentary front view similar to FIGURE 8, showing the cartridge 14 inserted part way into the drive module 18. The movable elements within the drive module 18 are all in the same operational positions that are shown in FIGURE 8. The cartridge 14 has been inserted to a point where the projections 87 and 88 on the lower end thereof are engaging the top surface of the horizontal portion of the eject member 171.

FIGURE 15 is a diagrammatic fragmentary front view which is similar to FIGURE 14, except that the cartridge 14 has been fully inserted, and various movable parts of the drive module 18 are in different operational positions. As one aspect of this, it will be noted that that the connector 83 of the drive module 18 is fully engaged with the connector 93 of the cartridge 14. Also, a front portion of the housing 85 of the cartridge 14 has been removed, so that only that a back portion 251 of the cartridge housing 85 is visible in FIGURE 15. To avoid unnecessary complexity in FIGURE 15, the circuitry and information storage structure provided within the cartridge 14 have been omitted.

With reference to FIGURE 15, it will be noted that the housing portion 101 of the drive module 18 has a small opening 261 provided vertically therethrough between the downward projection 61 and the recess 107. This opening can be used to manually operate of the release mechanism in the drive module, in a manner described in more detail later.

A brief discussion will now be provided of the operation of certain structure described above. In this regard, and as discussed above, the eject member 171 is capable of reciprocal vertical sliding movement within the drive module 18. As the eject member 171 moves vertically, the cam followers at the opposite ends of the central opening 186 can slide along the above-mentioned lower cam surfaces on the upper portions 131 (FIGURE 10) of the pawls 81 and 82. FIGURE 15 shows the lowest position of the eject member 171, in which the bottom surface of the eject member 171 engages the stop surfaces 156 (FIGURE 10) on the pawls 81 and 82. The stop

surfaces 156 prevent downward movement of the eject member 171 past the position shown in FIGURE 15.

The uppermost position of the eject member 171 is shown in FIGURES 8 and 14, where the cam followers at opposite ends of the central opening 186 each engage the recess 148 in a respective one of the pawls 81 and 82. The bevel surfaces 193 and 194 (FIGURE 11) on the eject member 171 each engage a respective pawl 81-82 at the corner between the surface portions 151 and 146, and portions of the bevel surface 187 each engage the bevel surface 154 on a respective pawl 81-82. This is the position that the eject member 171 will normally be in when no cartridge is inserted, as evident from FIGURE 8. The resilience exerted by the spring 119 on the pawls 81-82, in conjunction with the various bevel surfaces on the pawls 81-82 and the eject member 171, cause the cam followers on the eject member 171 to tend to be retained within the recesses 148 in the pawls.

It would theoretically be possible for the eject member 171 to move further upwardly from the position shown in FIGURE 8, until the pin 183 thereon engages the upper end of the slot 213, and/or until the upper ends of the edge portions 176-177 engage the surfaces 203. In particular, if an upward force were applied to the eject member 171 while it was in the position of FIGURE 8, the bevel surfaces 193-194 on the eject member 171 would pivot the pawls 81-82 inwardly against the urging of the spring 119 as the eject member 171 moved upwardly in response to the applied force. In the disclosed embodiment, however, the actuating pin 236 on the crank gear 229 moves the eject member 171 only up to the position which is shown in FIGURE 8, and there is no

other structure within the drive module which would move the eject member 171 any higher than the position of FIGURE 8 during normal operation.

With reference to FIGURE 14, when a cartridge 14 is inserted into the drive module 18 with the proper orientation, the guide rails 77 and 78 slide into the grooves 91 and 92 of the cartridge 14, and the cartridge 14 will reach the partially inserted position shown in FIGURE 14, where the projections 87 and 88 on the cartridge each engage the horizontal portion of the eject member 171. In response to the downward manual force which is still being exerted on the cartridge 14, the projections 87 and 88 will urge the eject member 171 to move downwardly. It will be noted that, with the crank gear 229 and the actuating pin 236 stopped in the angular position shown in FIGURE 8, the eject member 171 can move downwardly without engaging any portion of the drive mechanism 221, and in particular without contacting the actuating pin 236.

In response to this downward force being manually exerted on the cartridge 14, the portions of the beveled surface 187 at opposite ends of the opening 186, which define parts of the cam followers, cooperate with the angled surface portions 154 (FIGURE 10) on the pawls 81 and 82, so as to urge each of the pawls to pivot inwardly against the urging of the spring 119. As this occurs, the cam followers at the ends of the opening 186 slide down onto the surface portions 147 (FIGURE 10) of the pawls 81 and 82. This facilitates entry of the upper ends of the pawls 81 and 82 into the recesses 96 and 97 in the cartridge. The inclined surfaces 141-143 (FIGURE 10) at the upper ends of the pawls also help

guide the upper end of each pawl into a respective recess 96 or 97.

In response to further manual insertion of the cartridge, the cam followers at the ends of the openings 186 in eject member 171 will slide off the surface portions 147 of the pawls 81-82, and then down the inclined surface portions 158 (FIGURE 10) toward the surface portions 157. During this movement, the engagement of the upper ends of the pawls 81-82 with the recesses 96-97 in the cartridge 14 will eventually cause the pawls to be held in an inwardly pivoted position, in which the cam followers on the eject member 171 do not engage the surface portions 157 on the pawls 81-82. At this point, the eject member 171 can drop downwardly under the force of gravity until it engages the stop surfaces 156 (FIGURE 10) on the pawls 81-82.

Thereafter, further manual insertion of the cartridge 14 results in the connectors 83 and 93 moving into full engagement with each other, which is the operational position shown in FIGURE 15. In this position, and as shown in FIGURE 15, the edge portions of the recesses 96 and 97 in the cartridge reach a position in which they are aligned with the recesses 148 (FIGURE 10) in the pawls 81-82, and the pawls 81 and 82 can then pivot outwardly somewhat under the urging of the spring 119, to the positions which are shown in FIGURE 15. In FIGURE 15, interior surface portions of the cartridge housing adjacent each of the recesses 96-97 engage the surfaces 151 (FIGURE 10) of the pawls, thereby preventing upward movement of the cartridge relative to the pawls. The cartridge 14 is now releasably latched within the drive module 18 by the pawls 81 and 82.

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It should be noted that the pivot pins 116 and 117 for the pawls 81-82 are each disposed laterally outwardly from the point of engagement between each pawl and the associated edge of a recess 96 or 97. Thus, if the user attempts to manually withdraw the cartridge 14, the upward manual force exerted on the cartridge will tend to urge the pawls 81 and 82 to pivot outwardly, away from the connectors 83 and 93. Pivotal movement in this direction urges the recesses 148 in the pawls into tighter engagement with the housing of the cartridge, thus enhancing the locking engagement between the pawls 81-82 and the cartridge. accordingly, the cartridge 14 is securely held against manual withdrawal once it has been inserted to the position of FIGURE 15 and is latched in place there.

In order to remove the cartridge 14 in a normal manner, the user presses the eject button 27 (FIGURES 1 and 2) on the interface module 17, causing circuitry within the interface module 17 to send an electrical signal through the connectors 52 and 67 (FIGURES 2 and 4) to a circuit within the drive module 18. This circuit in the drive module 18 energizes the motor 223 of the drive mechanism 221 in a manner causing the crank gear 229 (FIGURE 8) to carry out one full revolution in a clockwise direction, and then stop again in the position shown in FIGURE 8.

As the crank gear 229 rotates, the actuating pin 236 thereon initially moves downwardly and to the left in FIGURE 8, and thereby moves under the downwardly facing surface 181 on the eject member 171. The eject member 171 and the surface 181 thereon will, of course, be physically lower at this time than the position shown in

FIGURE 8. As the crank gear 229 continues to rotate, the actuating pin 236 will begin to move upwardly and, due to engagement with the surface 181, will force the eject member 171 to also move upwardly.

5 As the eject member 171 is moved upwardly, the cam followers at the ends of the central opening 186, including the bevels 193 and 194 (FIGURE 11), will engage the inclined surfaces 158 (FIGURE 10) on the pawls 81 and 82, and thus force the pawls 81 and 82 to pivot inwardly
10 against the urging of the spring 119. This effects disengagement of each pawl 81-82 from the associated recess 96 or 97 in the cartridge 14. As the eject member 171 continues moving upwardly, it will engage the projections 87 and 88 on the lower end of the cartridge
15 14, and then the cartridge 14 will be moved upwardly by the eject member 171 as the eject member continues moving upwardly. As the eject member 171 moves the cartridge 14 upwardly in relation to the drive module 18, the pawls 81-82 will exit the recesses 96-97 in the cartridge 14,
20 and the connectors 83 and 93 will be separated from each other.

With reference to FIGURE 8, rotation of the crank gear 229 will cause the actuating pin 236 to eventually reach the vertically highest position along its path of travel, in which it has raised the eject member 171 to
25 the position shown in FIGURE 8. After that, the actuating pin 236 will move rightwardly and downwardly out of engagement with the surface 181, back to the position which is shown in FIGURE 8.

30 As the pin 236 leaves engagement with the eject member 171, with the eject member 171 in the position shown in FIGURE 8, the cam followers at the ends of the

central opening 186 in the eject member 171 will each be engaging the recess 148 in a respective one of the pawls 81-82. Because the ridges 91-92 increase the effective thickness or height of the eject member in the region of the cam followers, the cam followers will only be able to partially enter the recesses 148 in the pawls. Consequently, the angled surfaces 154 on the pawls will each be engaging the bevel surface 187 on the eject member, so that the eject member 171 is ready to move out of engagement with the recesses 148 in the pawls in response to the application of a downward force to the eject member.

In this operational position, the combined weight of the eject member 171 and the cartridge 14 supported on it does not generate enough of a downward force to cause the cam followers on the eject member 171 to pivot the pawls 81 and 82 inwardly. Thus, the eject member 171 will tend to remain in the position shown in FIGURE 8, where it supports the cartridge in a position with the connectors 83 and 93 disengaged and slightly spaced from each other. The user can then manually lift the cartridge directly out of the drive module 18. Since the connectors 83 and 93 are fully disengaged, and the pawls 81-82 are no longer engaging the cartridge, there is no significant friction or other force tending to retain the cartridge 14 within the drive module 18. Thus, the user can lift the cartridge 14 out of the drive module 18 with one hand, without any need to use the other hand to hold down the cradle 13 while forcibly separating the cartridge 14 from the drive module 18.

When a cartridge 14 has been fully inserted to the position shown in FIGURE 15, and as was discussed above

in association with FIGURE 15, an attempt by a user to manually withdraw the cartridge from the drive module 18 will not be successful, because it will tend to pivot the pawls 81-82 in a direction which increases the effective locking force. This is why the normal mode of removing the cartridge 14 is to press eject button 27 (FIGURES 1-2), in order to energize the drive mechanism 221 and permit it to automatically release the pawls 81-82 and then partially eject the cartridge 14, so that the connectors 83 and 93 become separated. However, there may be rare circumstances in which it would be desirable to be able to manually remove the cartridge 14 from the drive module 18 without relying on the drive mechanism 221.

For example, the drive module 18 receives operating power from the interface module 17, and can be detached from the interface module 17. Thus, if the drive module 18 has a cartridge inserted but is not currently coupled to an interface module, the drive mechanism 221 has no electric power, and cannot be used to release and eject the cartridge 14. Another unlikely but possible scenario is that, after extensive use of the drive module 18 over a long period of time, there may be some form of failure within the drive mechanism 221 which prevents it from operating, even when the drive module 18 is supplied with power. In this situation, a user may need to return the drive module 18 to a service center for repair, but would not want to send the cartridge 14 to the service center, particularly where the cartridge 14 included confidential or other sensitive data. For these reasons, the drive module 18 includes provisions for effecting a manual

release and ejection of a cartridge in rare or emergency circumstances.

More specifically, and with reference to FIGURE 8, an elongate release element 291, which is shown diagrammatically in FIGURE 8, can be manually slidably inserted through the manual release opening 261 in the housing of the drive module, until its upper end engages the underside of the eject member 171. By then manually applying an upward force to the release element 191, it will force the eject member 171 to move upwardly from its lowermost position (FIGURE 15) to its uppermost position (FIGURE 8).

As the eject member 171 carries out this upward movement, it disengages the pawls 81-82 from the cartridge 14 and moves the cartridge 14 upwardly so as to separate the connectors 83 and 93, in a manner identical to that discussed above for movement of the eject member 171 by the drive mechanism 221. It will be noted that, even if the crank gear 229 has stopped in a position other than the normal stop or idle position shown in FIGURE 8, the eject member 171 is configured so that it is always capable of upward movement relative to the actuating pin 236. Once the eject member 171 has been manually moved upwardly to the position of FIGURE 8 by the release element 291, the release element 291 can be withdrawn from the opening 261, and the cartridge 14 can be manually lifted out of the drive module 18.

FIGURE 16 is a diagrammatic fragmentary front view of an alternative embodiment of the drive module of FIGURE 3. FIGURE 16 is similar to a portion of FIGURE 8 depicted in an enlarged scale, and equivalent components are designated with the same reference numerals. The

embodiment of FIGURE 16 is generally the same as the embodiment of FIGURE 8, except for the differences discussed below.

FIGURE 16 depicts an additional mounting plate 311, which is fixedly secured to the mounting plate 222 (FIGURE 8) for the motor 223 (FIGURE 8). A position detect switch 312 of a known type is supported on the mounting plate 311, and has an operating member 316 which is pivotally supported by a pivot pin 317. The operating member 316 can pivot counterclockwise about the pivot pin 317 in FIGURE 16, from the advanced position shown in FIGURE 16 to a retracted position where it is disposed within the housing of the switch 312. The operating member 316 is resiliently urged toward its advanced position by a not-illustrated spring. As the operating member 316 pivots from the advanced position to the retracted position, the switch is actuated.

The housing of the switch 312 is disposed closely adjacent the peripheral edge of the crank gear 229. In its advanced position, the operating member 316 projects radially inwardly adjacent a side surface of the crank gear 229. The terminals of the switch 312 are electrically coupled to the above-mentioned circuit which controls the motor 223.

The operation of the embodiment of FIGURE 16 is generally similar to the operation of the embodiment of FIGURES 1-15, except as follows. More specifically, as the motor 223 rotates the gear 229 in a clockwise direction in FIGURE 16, the actuating pin 236 thereon will eventually reach the angular position which is shown in broken lines in FIGURE 16. As the actuating pin 236 moves beyond this position, it will engage the operating

member 316 and pivot the operating member from its advanced position to its retracted position, thereby actuating the switch 312. The circuit which controls the motor 223 will then know the current angular position of the pin 236, and in particular will know that the pin has just reached the beginning of a range of positions in which, if the gear 229 is stopped with the pin 236 disposed within that range of positions, the pin 236 will not be able to interfere with reciprocal movement of the eject member 171. Consequently, the circuit can turn the motor 223 off as soon as the switch 312 is actuated, or a short predetermined time interval after the switch 312 is actuated, in order to stop the gear 229 in a position in which the pin 236 is within the range of positions where it cannot interfere with movement of the eject member 171.

FIGURE 17 is a diagrammatic fragmentary perspective rear view of a portion of a drive module 410, which is an alternative embodiment of the drive module 18 shown in FIGURES 3 and 5. FIGURE 18 is a diagrammatic perspective view showing rear and bottom sides of an eject member 412 which is a component of the drive module 410 of FIGURE 17, and which is an alternative embodiment of the eject member shown at 171 in FIGURES 11 and 12. The drive module 410 and eject member 412 are similar to the drive module 18 and eject member 171 described previously, except for the differences described below.

More specifically, as discussed above in association with FIGURES 8, 14 and 15, the housing of the drive module 18 has a manual release opening 261. In contrast, the drive module 410 does not have the opening 261, but instead has a vertical slot 414 provided through an

approximately vertical wall of the housing that is located near the rear side of the eject member 412. In FIGURE 17, the eject member 412 can be seen through the slot 414. As discussed above in association with the drive module 18, the opening 261 in that drive module is positioned so that it is accessible only when the drive module 18 is disengaged from and physically separated from the interface module 17. With reference to the drive module 410, it will be noted that the slot 414 is also positioned so that it is accessible only when the drive module 410 is disengaged from and physically separated from the interface module 17.

As shown in FIGURES 17 and 18, the eject member 412 has a cylindrical opening 421 extending horizontally through a vertical wall thereof. The opening 421 is located approximately halfway between the outer edges 176 and 177 of the eject member 412. The eject member 412 also has on the rear side thereof a rearwardly-projecting semicircular ridge 422, which is concentric to and adjacent the opening 421, and which is located on the upper side of the opening 421. As the eject member 412 moves from its retracted position to its eject position, the opening 421 in the eject member 412 moves from a position aligned with a lower portion of the slot 414 to a position aligned with an upper portion of the slot 414.

In the embodiment of FIGURES 17 and 18, a manual release is effected in the following manner. It is assumed for purposes of discussion that the drive module 410 has therein a cartridge 14 (which is not visible in FIGURES 17-18), and has been physically detached from the interface module 17 so that the slot 414 is accessible. An elongate release element (which is not illustrated but

is similar to the release element shown at 291 in FIGURE 8) is inserted into the slot 414 so as to engage the underside of the ridge 422 adjacent the opening 421. Although it is sufficient if the end of the release element engages the ridge 422 but does not extend into the opening 421, it is preferable that the end of the release element extend into the opening 421.

Since the eject member 412 will be in its retracted position, the opening 421, the ridge 422 and the release element will be near the lower end of the slot 414. The release element is then moved manually upwardly within the slot 414, so that the eject member 421 is moved manually upwardly from its retracted position to its eject position. This effects a release and ejection of the cartridge in the manner described above in detail for the embodiment of FIGURES 1-15. The release element is then manually withdrawn from the slot 414. Since the opening 421 and the ridge 422 are approximately halfway between the outer edges 176 and 177 of the eject member 412, the upward force exerted manually on the eject member 412 is applied in approximately the center of the eject member 412. Accordingly, the entire eject member 412 tends to move smoothly upwardly without being urged by the applied force to rotate about some axis parallel to the axis of the opening 421.

The present invention provides a number of technical advantages. One such technical advantage is that the removable data storage cartridge is reliably and releasably locked within the drive after it is inserted, and remains locked there until an eject mechanism within the drive is actuated. A related advantage is that, when the cartridge is releasably locked within the drive,

exertion of a manual withdrawal force on the cartridge has the effect of increasing the effective locking force.

According to another advantage, the eject mechanism is a motor-powered mechanism. A related advantage is that this eject mechanism is simple and involves a minimum number of parts, and is thus relatively inexpensive. A further advantage is that the motor which drives the eject mechanism rotates in only a single direction. Still another advantage is that the eject mechanism effects not only a release of the locking mechanism which releasably holds the cartridge, but also effects sufficient movement of the cartridge to achieve full disengagement of the connectors which electrically couple the cartridge and the drive.

Still another advantage is that the eject mechanism resets the releasable locking arrangement to a state in which it is ready for insertion of another cartridge. A further related advantage is that a single movable eject member is configured to effect multiple functions associated with cartridge ejection, including release of the locking mechanism, cartridge movement that disengages the connectors, and resetting of the locking mechanism so it is ready for insertion of another cartridge. Yet another advantage is the eject member can move freely when the eject mechanism is disabled, for example so that manual insertion of a cartridge is possible without any interference from the eject mechanism, and without any need to energize the eject mechanism.

Although selected embodiments have been illustrated and described in detail, it will be understood that various substitutions and alterations are possible

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PATENT APPLICATION

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without departing from the spirit and scope of the
present invention, as defined by the following claims.

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